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a variable cycle third stream fan flow stream deployed. That is, the two dimensional secondary nozzle 12 may be a third stream exhaust nozzle which regulates a third flow stream selectively sourced from the fan section 101 and/or the compressor section 102. Notably, performance of engine 100 can be affected by regulating the secondary flow S by varying the two dimensional secondary nozzle 12.

In this regard, reference is made to FIG. 3, which depicts nozzle assembly 10 and the incorporated sliding door 16. As shown in FIG. 3, sliding door 16 is configured to be translated across a passage 18 of the generally planar secondary nozzle 12 and thereby influence gas directed along secondary flow path 26.

In at least one embodiment, the nozzle assembly 10 is a third stream exhaust nozzle that is operative to regulate gas accelerated by a tertiary fan 101B (e.g., a tip fan located radially outboard of a fan stage; FIG. 1). A third stream nozzle facilitates a variable engine cycle. By way of example, by closing the third stream nozzle area, the third stream duct experiences increased backpressure and the fan air normally flowing into the third stream duct diverts into the secondary/primary flow stream. Notably, the flow streams communicate just aft of the fan section 101. A third stream splitter, which can be located several inches aft of the fan, for example, leaves a large enough area for effective flow stream communication. However, in other embodiments, a nozzle assembly can be used for varying the flow characteristics of gas directed along one or more other gas paths.

The sliding door 16 is configured to be variably positioned along a range of positions between a full open position, at which the generally planar secondary nozzle 12 exhibits a maximum exit area, and a full closed position, at which the generally planar secondary nozzle 12 exhibits a minimum exit area. As the sliding door 16 is variably positioned, gas directed along secondary flow path 26 is regulated.

In the embodiment of FIG. 3, the sliding door 16 exhibits a low section area relative to a direction of travel associated with gas directed along secondary flow path 26. Such a configuration and orientation tends to result in a low actuation load, i.e., the load required to be overcome for positioning of the sliding door 16. In this regard, the nozzle assembly 10 also incorporates an actuator 20 that engages sliding door 16. The actuator 20 is attached to the sliding door 16 and is configured to operatively translate the sliding door 16 in both a fore and aft direction, generally parallel to the engine axis A as indicated by arrow D. The actuator 20 can be an air motor driven direct actuated ball screw ram, direct actuated hydraulic ram, and air or hydraulic driven mechanisms. Actuator 20 may be singular or a plurality of synchronized actuators. For example, the actuator 20 includes air motor driven direct actuated ball screw rams (such as linear motion cylindrical actuators or rotary motion actuators), synchronized via flex drive cables (a commonly used actuation configuration in various commercial nacelle reverser cowlings). The actuator 20 can be located unobtrusively in an area 22 of the nozzle assembly 10 between the secondary duct 110 and the primary duct 112.

In some embodiments, the nozzle assembly 10 may incorporate a pressurized plenum. Such a pressurized plenum can be configured to provide pressure balancing to the nozzle assembly thereby reducing actuation loads. If the loads are predicted to be reacted primarily by the tracks, a plenum may not be required. However, when a plenum is utilized (such as in association with area 22 in this embodiment), the plenum can be a direct acting plenum placed, for example, on the forward face of the sliding door 16. Alternatively, a remote balance chamber can be utilized.

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The nozzle assembly 10 also incorporates a rail 24 for the sliding door 16. The rail 24 facilitates the translation of the sliding door 16. In particular, the rail 24 provides a track on which the sliding door 16 is translated. The rail 24 also is configured to provide alignment and structural stability to the sliding door 16. In at least one embodiment, more than one rail 24 may be utilized. In other embodiments, the rail 24 includes one or more bearings to facilitate a smoother translation of the sliding door 16 along the rail 24. In yet another embodiment, the tracks of the rail 24 can be embedded in the fixed structure ahead of the sliding door 16, and/or along sides of the sliding door 16, such that sliding door 16 is cantilevered aft and the tracks are hidden from the flowpath.

The nozzle assembly 10 also incorporates a plurality of stiffening ribs 28 to control deflection of the sliding door as the sliding door 16 is variably opened and closed. For example, as the sliding door 16 is variably closed, pressure increases within the secondary duct 110. The plurality of stiffening ribs 28, located behind the interior wall of the secondary duct 110, reduces deflection of area 22. The plurality of stiffening ribs 28 also provides structural support to the sliding door 16 as the door translates across the passage 18.

As shown in FIG. 4, the actuator 20 is connected to the sliding door 16 and is configured to operatively translate the sliding door 16 in both a fore and aft direction, as indicated by arrow D, thus varying the flow through the generally planar secondary nozzle 12 of the nozzle assembly 10. In operation, the sliding door 16 is variably opened and closed, by translating in both a fore and aft direction. In other embodiments, more complex motion of the sliding door can be used. Regardless of the particular motion involved, positioning of the sliding door 16 varies the nozzle assembly 10 and thereby affects one or more of various engine performance characteristics.

It should be emphasized that the above-described embodiments are merely possible examples of implementations set forth for a clear understanding of the principles of this disclosure. Many variations and modifications may be made to the above-described embodiments without departing substantially from the spirit and principles of the disclosure. By way of example, in some embodiments, a sliding door can be configured to alter a nozzle throat asymmetrically in order to affect yaw vectoring of the flow. In some embodiments, this can be accomplished by the use of differential actuation of multiple actuators. All such modifications and variations are intended to be included herein within the scope of this disclosure and protected by the accompanying claims.

The invention claimed is:

1. A nozzle assembly for a gas turbine engine, the nozzle assembly comprising:

- a door operative to selectively increase and decrease an effective size of the nozzle exit area;
- a secondary flow duct with a two dimensional secondary nozzle to communicate a secondary flow therethrough;
- a primary flow duct with a two dimensional primary nozzle to communicate primary flow therethrough, said two dimensional primary nozzle adjacent to said two dimensional secondary nozzle;
- said door axially slidable relative to a passage adjacent a secondary flow path for said secondary flow and a primary flow path for said primary flow, said door axially slidable relative to a passage in communication with said secondary flow path; and